

(Following Paper ID and Roll No. to be filled in your  
Answer Books)

Paper ID : 151410

Roll No. 

|  |  |  |  |  |  |  |  |  |  |
|--|--|--|--|--|--|--|--|--|--|
|  |  |  |  |  |  |  |  |  |  |
|--|--|--|--|--|--|--|--|--|--|

**B.TECH.**

**Theory Examination (Semester-IV) 2015-16**

**HEAT TRANSFER**

*Time : 3 Hours*

*Max. Marks : 100*

**Section-A**

**1. Attempt all parts of the following : (2×10=20)**

- (a) Explain Kirchhoff's law
- (b) Define Black body & Gray body
- (c) Define Natural and forced convection
- (d) Define Capacity and economy of evaporator
- (e) Explain Dropwise and Filmwise condensation
- (f) Explain physical significance of Nusselt No, Prandtl No, Grashoff No, Stanton No.
- (g) Discuss in brief about molten metal.
- (h) What do you understand by overall heat transfer coefficient?
- (i) What are fins? Write its application?
- (j) Explain the fouling factor.

## Section-B

### 2. Attempt five parts of the following. (5×10=50)

- (a) What is optimum thickness of insulation? Derive the critical thickness of insulation of Cylinder.
- (b) An industrial wall is constructed of 20cm thick fireclay,  $k=1\text{ W/m}^\circ\text{C}$ . This is covered on outer surface with a 3 cm layer of insulating material,  $k=0.075\text{ W/m}^\circ\text{C}$ . The innermost surface is at  $940^\circ\text{C}$  & the outermost at  $40^\circ\text{C}$ . Calculate the steady rate of heat transfer through the wall in  $\text{W/m}^2$  & the temperature of the interface between the fireclay & the insulating material.
- (c) The outer surface temperature of a pipe (radius = 0.1 m) is 400 K. The pipe is losing heat to atmosphere, which is at 300K. The film HTC is  $10\text{ W/m}^2\text{K}$ . To reduce the rate of heat loss, the pipe is insulated by a 50 mm thick layer of asbestos,  $k = 0.5\text{ W/mK}$ . Calculate the percentage reduction in rate of heat loss.
- (d) A horizontal steam pipe 20m long, 50mm internal diameter, 60mm outside diameter losses 3.5kw heat to the surrounding at 310k. The pipe carries steam at 500k given that the convective heat transfer coefficient  $h_c=1.65(\Delta T)^{-0.25}\text{ W/m}^2\text{ k}$  and the Stefan Boltzmann constant= $5.67*10^{-8}\text{ W/m}^2\text{ k}^{-4}$ . Find the emissivity of the bare surface of the pipe.

- (e) Oil at  $120^{\circ}\text{C}$  is used to heat water at  $30^{\circ}\text{C}$  in a 1-1 con-current shell and tube heat exchanger. The available heat exchanger area is  $A_1$ . The exit temperature of the oil and the water streams are  $90^{\circ}\text{C}$  and  $60^{\circ}\text{C}$  respectively. The con-current heat exchanger is replaced by 1-1 counter current heat exchanger having heat exchange area  $A_2$ . If the exit temperature and the overall heat transfer coefficient are same. What is the ratio of  $A_1$  to  $A_2$ ?
- (f) A double pipe heat exchanger is to be designed to heat 4 kg/s of a cold feed from  $20$  to  $40^{\circ}\text{C}$  using a hot stream available at  $160^{\circ}\text{C}$  and a flow rate of 1 kg/s. The two streams have equal specific heat capacities and the overall heat transfer coefficient of the heat exchanger is  $640 \text{ W/m}^2 \text{ K}$ . What is the ratio of heat transfer area for con-current and counter current modes of operations?
- (g) Explain hydrodynamic boundary layer & thermal boundary layer.
- (h) Prove by dimensional analysis that  $\text{Nu} = f(\text{Re}, \text{Gr})$

### Section-C

Attempt two parts of the following.

(2×15=50)

3. (i) In a concurrent heat exchanger, an oil stream is cooled from  $450\text{K}$  to  $410\text{K}$  by water inlet & outlet temp<sup>s</sup> of  $300\text{K}$  &  $350\text{K}$  respectively. The exchanger consists of a number of tubes of 1m length each. It is now desired to cool the oil to  $390\text{K}$  (instead of  $410\text{K}$ ) while

(3)

P.T.O.

maintaining the flow rate of oil, flow rate of water, inlet temps of oil & water & number of tubes at the same values as before. Calculate the length of each tube required for this purpose. Assume the physical properties remain unchanged.

- (ii) An aqueous solution of a solute is concentrated from 5% to 20% (mass basis) in a single effect short tube evaporator .The feed enters the evaporator at a rate of 10 kg/s & at a temp of 300K steam is available at a saturation pressure of 1.3 bar. The pressure in the vapor space of the evaporator is 0.13 bar & the corresponding saturation temperature of steam is 320K. If the overall HTC is 5000w/m<sup>2</sup>K. Calculate steam economy & heat transfer surface area.

Data:

|                                    | Enthalpy<br>(KJ/kg) | Heat of vaporization<br>(KJ/kg) |
|------------------------------------|---------------------|---------------------------------|
| Saturated steam<br>(1.3 bar, 380K) | -----               | 2000                            |
| Saturate steam<br>(0.13bar, 320K)  | 2200                | -----                           |
| Feed (5%, 325 K)                   | 80                  | -----                           |
| Concentrated liquor<br>(20%, 325K) | 400                 | -----                           |

BPR is 5K

(4)

4. (i) Saturated steam at  $6.9 \times 10^4$  Pa pressure and  $90^\circ\text{C}$  saturation temperature condenses on a vertical pipe of 0.025m OD and 0.3m length. The average condensing HTC on the tube is  $12000 \text{ W/m}^2\text{K}$ .

**Data:**

Outside surface temperature of the pipe =  $86^\circ\text{C}$

Enthalpy of saturated steam =  $2659 \text{ kJ/kg}$

Enthalpy of condensate =  $375 \text{ kJ/kg}$

Viscosity of condensate at the film temperature  
=  $3.24 \times 10^{-4} \text{ pa s}$

Assume the flow of the condensate is laminar.

Calculate the rate of steam condensation.

Check whether the flow is laminar.

- (ii) Explain capacity & economy of evaporator. Show by sketch, the various methods of feeding arrangement in triple effect evaporators. Discuss their relative merits & demerits.
5. (i) A horizontal steam pipe 220m long, 50mm ID, 60mm OD loses  $13.5 \text{ kW}$  heat to the surroundings at  $310 \text{ K}$ . The pipe carries steam at  $500 \text{ K}$ . Given that the convective heat transfer coefficient  $h_c = 1.65 (\Delta T)^{0.25} \text{ W/m}^2\text{K}$  & Stefan - Boltzmann constant,  $\sigma = 5.67 \times 10^{-8} \text{ W/m}^2 \text{ K}^4$ . Find the emissivity of the bare surface of the pipe.

- (ii) In a 1- 1 counter flow shell & tube heat exchanger, a process stream (  $C_p=4.2\text{kJ/kgK}$ ) is cooled from 450K to 350K Using water ( $C_p=4.5\text{ kJ/kgK}$ ) at 300K. The process stream flows on the shell sides at a rate of 1 kg/s & the water on the tube side at a rate of 5 kg/s. If HTC on the shell & tube sides are  $1500\text{ W/m}^2\text{ K}$  &  $1500\text{ W/m}^2\text{K}$  respectively.

Determine:-

- I. The required heat transfer area
- II. By what factor will the required area change if the flow is co-current?

